Role of vegetation density on wildland fire



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Motivation Numerical configuration Methodology Results Conclusions



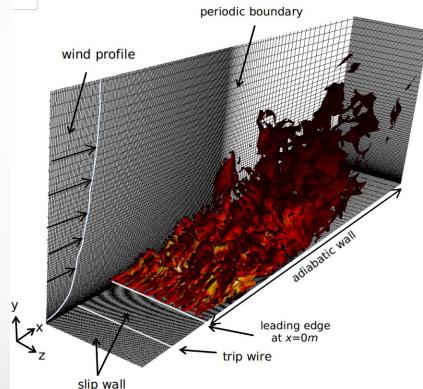
- Is Froude number (*Fr*), Byram number (*Nc*) or other dimensionless number enough to describe fire behaviour?
- Lack of understanding of the effect of vegetation density (*CdLAI*) on
 a. Fire regime transition from plume dominated to wind driven.
 b. Heat transfer mechanism from radiative to convective heat transfer to ignite unburnt fuel.
- Lack of understanding of the configuration space {*Nc, CdLAI*} effect on fire behaviour.

Sub-grid small-scale flame models is essential for large- or gigascale fire modelling - *e.g.*, coupled fire-atomosphere modelling

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(Verma S. and Trouve A., 2018)



Visualization of the configuration using instantaneous iso-contour of Q-criterion at $2(s^{-2})$, clipped to $x_{max} = 2.5m$, $y_{max} = 1m$.

Solver: Simplified ForestFireFoam (FFF) developed from FireFoam at M2P2 Lab.

Features of FFF:

Multiphase turbulent combustion modelling with sub-models improvement for:

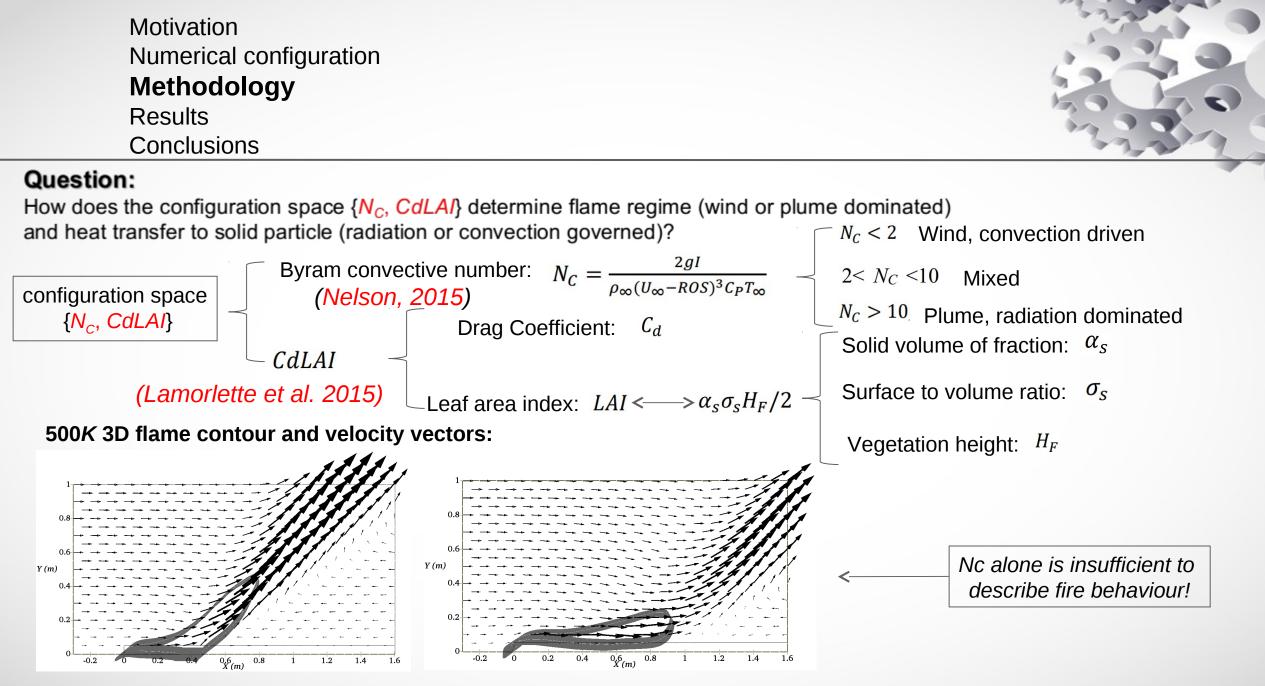
a. Pyrolysis; b. Evaporation; c. Char oxidation/Smoldring

- d. Turbulent combustion:
- WALE;
- EDC;

e. Radiation: fvDOM

Utilities for pre- or post-processing:

- blockMesh; setFields; topoSet; funkySet, swak4Foam, paraview etc.



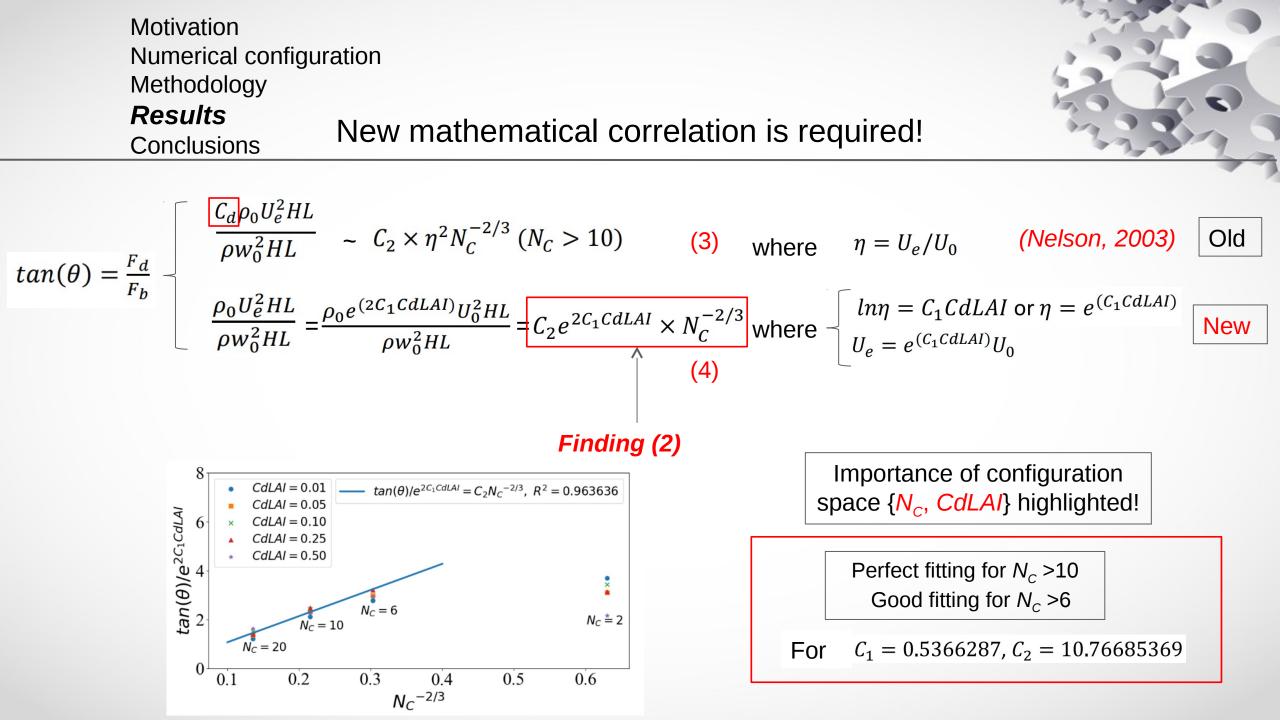
Nc=20, CdLAI=0.01

Nc=20, CdLAI=0.5

Motivation Numerical configuration Methodology Results Conclusions Model correlations Fuel (Hu et al. 2013 $tan(\theta) = C \times (\frac{2T_0}{T_f})^{1/5} \times Ka^{-1/5}$ (1) Flame tilt angle Heptane, Ethanol, U_0 Tang et al. 2015) and Acetone definition H_{flame} $tan(\theta) = \begin{cases} C_1 \times \alpha^{1/2} N_C^{-1/3} (N_C < 10) \\ C_2 \times \eta^2 N_C^{-2/3} (N_C > 10) \end{cases}$ (2) Long leaf pine, (Nelson, 2003) slash pine litter, $\uparrow H_F$ etc. W_{flame} 90 CdLAI = 0.01 $tan(\theta) = 5.13328 N_c^{-1/3}, R^2 = 0.97$ CdLAI = 0.01CdLAI = 0.05CdLAI = 0.05CdLAI = 0.10 80 70 60 60 CdLAI = 0.10CdLAI = 0.25 $tan(\theta)$ CdLAI = 0.25CdLAI = 0.50CdLAI = 0.50 $N_c = 2$ fitting? $N_C = 6$ $N_{C} = 10$ × $N_{c} = 20$ 0 0.3 0.8 0.4 0.5 0.6 0.7 50 2 10 20 6 $N_{C}^{-1/3}$ N_C $N_C = \frac{2gI}{\rho_{\infty}(U_{\infty} - ROS)^3 C_P T_{\infty}}$ $R_i = g\Delta T_f D / T_f U_0^2$ $tan(\theta) = C \times \left[2 \times \frac{\rho_0 C_{p,0} T_0 U_0^3}{2gI} \times \frac{U_0^2 T_f}{g(T_f - T_0)D} \times \frac{T_0}{T_f}\right]^{1/5} = C \times \left(\frac{2T_0}{T_f}\right)^{1/5} \times [K_a]^{-1/5}$ $K_a = N_C \times R_i$

Interestingly, $Ka^{-1/5} \sim N_c^{-1/3}$, equation (1) and (2) are essentially the same, only applicable to $N_c < 10$, role of CdLAI?

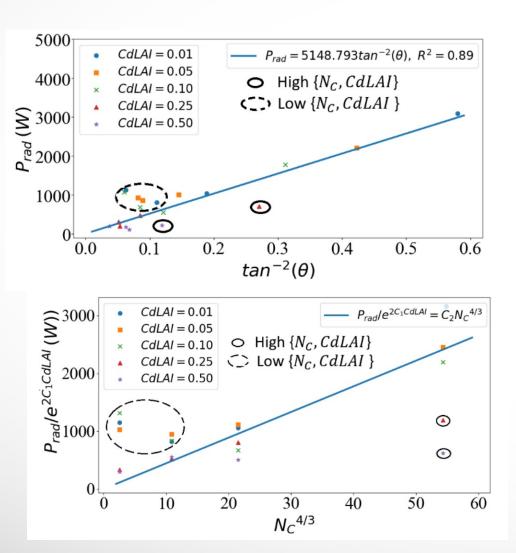
Finding (1)



Motivation Numerical configuration Methodology **Results**

New mathematical correlation is required!

Conclusions



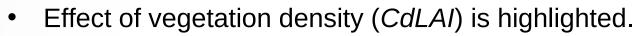
Definition of radiative heat power: $P_{rad} = \int Q_{rad}^s dv / C dLAI$

 $P_{rad} = 5148.793 tan^{-2}(\theta)$ with $R^2 = 0.89$

 $P_{rad} = C_2 e^{2C_1 C d L A I} N_C^{4/3}$ With $C_1 = -1.0732574$ and $C_2 = 44.41481$



Motivation Numerical configuration Methodology Results **Conclusions**



- Flame tile angle ~ directly related to CdLAI, inversely related to N_c .
- Radiative heat power ~ directly related to N_c , inversely related to *CdLAI*.
- Fitting with experimetanl data needed to validate proposed model correlations.

Thank You